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PATENT APPLICATION

BY

TRAVIS R. PIEHL & KERMIT A. WOLD

FOR

PROPORTIONAL DIRECTIONAL CONTROL VALVE WITH A MAGNETIC POSITIONING SENSOR

Howard B. Rockman Reg. No. 22,190 Barnes & Thornburg One North Wacker Drive, Suite 4400 Chicago, IL 60606-2809 Phone (312) 357-1313 Facsimile (312) 759-5646 Direct Line (312) 214-4812

PROPORTIONAL DIRECTIONAL CONTROL VALVE WITH A MAGNETIC POSITIONING SENSOR

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Cross-Reference to Related Applications

[0001] This application claims priority under 35 U.S.C. section 119(e) from copending U.S. *Provisional* Patent Application No. _______, filed September 11, 2000, naming Travis Raymond Piehl as inventor, and titled "PROPORTIONAL DIRECTIONAL CONTROL VALVE WITH A MAGNETIC POSITIONING SENSOR".

Field of the Disclosure

[0002] This invention is directed to proportional control valves, and more particularly, to a proportional directional control valve with a magnetic positioning sensor.

Background

[0003] Use of proportional control valves is widespread in many industries. For example, a proportional control valve may be used to control a position of a machine tool such as a saw requiring accurate positioning and repeatability of that position within a small tolerance. Proportional control valves may also be used in applications where speed control may be required, for example, to maintain a precise speed of a web traveling through a printing press having varying roll sizes. Similarly, proportional control valves may be used in applications requiring force or pressure control such as pressure control for cylinders being utilized to lift or hold a load.

[0004] In an effort to optimize proportional control valve performance. manufacturers have attempted (1) to minimize the number of moving and/or contacting components of the valve, (2) to minimize affects of temperature, fluid viscosity, contaminates, etc. on the components of the valve, and (3) to optimize valve operating parameters. Optimizing the valve operating parameters may include increasing the response time of the valve (i.e., the time required for the valve output to reach the new level when the valve input current is stepped in amplitude), improving repeatability (i.e., the valve's ability to return to the same output flow on repeated trials), reducing hysteresis (i.e., the difference in valve input current required to produce the same output as the valve is slowly cycled); and reducing deadband (i.e., the region of no response), to name a few. [0005] One current design used to minimize the number of components in contact with one and another, and to optimize valve operating parameters of a proportional control valve, includes the use of a well-known linear variable differential transformer (LVDT). In general, the LVDT is composed of a two secondary coils placed symmetrically on either side of a primary coil contained within a hollow cylindrical shaft, and a moveable solid magnetic core. When coupled to a linearly moveable valve spool, displacement of the magnetic core relative to the hollow cylindrical shaft causes the mutual inductance of each secondary coil to vary relative to the primary coil. As a result, the LVDT provides an electrical output proportional to a position of the magnetic core and therefore the position of the coupled valve spool. Accordingly, subsequent adjustments based on the electrical output can made to the position of the valve spool and a

resulting fluid flow through the valve spool. Although effective for providing an electrical output proportional to a position of the valve spool, the LVDT is relatively costly.

[0006] Typically, set-up, initializing and overall operational control of a proportional control valve is accomplished using individual electronic components that are remotely located from the valve; they are not resident in the valve housing. Additionally, current designs utilizing such individual electronic components often require manual potentiometer adjustments and manual jumper reconfigurations for set-up and control of the proportional control valve. While the use of individual electronic components simply reflects an inefficient use of space and of currently offered integrated circuit technology, the manual intervention required to maintain, set-up and control typical proportional control valves contributes adversely to the overall cost of operating and maintaining the proportional control valve.

Summary of the Invention

[0007] A proportional directional control valve with a magnetic sensor is disclosed herein. In one embodiment, the proportional directional control valve includes a housing having a first portion and a second portion where the first portion has an interior chamber, an electronic controller mounted within the interior chamber, at least one solenoid assembly electrically coupled to the electronic controller where the solenoid assembly is disposed in the second portion and where a drive pin of the solenoid assembly is reciprocally moveable responsive to a current

provided by the electronic controller, a valve assembly disposed in the second portion where the valve assembly includes a linearly moveable valve element operatively connected to the solenoid assembly where a position of the valve element is responsive to a position of the drive pin, a magnetic assembly operatively connected to the valve element where the magnetic assembly provides a magnetic field responsive to the position of the valve element, and a magnetic positioning sensor assembly coupled to the electronic controller where the magnetic positioning sensor detects a change in the magnetic field and generates an output voltage proportional to the change in the magnetic field. The current to the solenoid assembly is adjusted by the electronic controller responsive to a comparison of the output voltage and a command electrical input signal. The first portion and the second portion are easily separable enabling the first portion to be replaced independently from the second portion and enabling the second portion to be replaced independently from the first portion. [0008] The magnetic positioning sensor assembly preferably includes a linear Hall-effect sensor. The magnetic assembly includes a non-magnetic housing having a passageway extending axially through a portion of the non-magnetic housing, a first magnet disposed in the passageway, a second magnet disposed in the passageway where a south pole of the second magnet is separated from a north pole of the first magnet by a non-magnetic spacer in the passageway, and a retaining ring disposed in the passageway holding the first and second magnets and the non-magnetic spacer in a fixed position. The non-magnetic

housing preferably includes a beryllium copper housing and the first and second magnets preferably include a samarium cobalt material.

[0009] The proportional directional control valve also includes an operator interface port coupled to the electronic controller to enable values for at least one operating parameter of the proportional directional control valve to be downloaded from a set-up device to the electronic controller. The set-up device preferably includes a computer. The operator interface port preferably includes a universal serial bus port. The parameters include one of more of a solenoid coil voltage, a command input voltage, a command input current, solenoid enable switch, a solenoid null parameter, a solenoid gain parameter, a solenoid acceleration parameter, a solenoid deceleration parameter a dither frequency parameter and a solenoid dither amplitude parameter. The proportional directional control valve further includes a removable protective cap to allow operator access to the operator interface port.

[0010] The proportional directional control valve may also include a wireless transceiver coupled to the electronic controller to enable values for the at least one operating parameter of the proportional directional control valve to be downloaded from the set-up device to the electronic controller. A main connector coupled to the electronic controller is also provided. The main connector is configured to receive electrical power and to provide the electrical power to the electronic controller, to receive the command electrical input signal where the command electrical input signal is processed by the electronic

controller to drive the solenoid assembly, to provide an alarm output and to enable current monitoring.

[0011] The electronic controller is mounted to an electronic board in the interior chamber and the proportional directional control valve further includes an electronic potting material disposed on the electronic board where the electronic potting material provides a protective barrier for the electronic board.

[0012] The valve assembly includes a chamber disposed in the valve assembly where a source of fluid is selectively connected to an input channel of the chamber, and an output channel in the chamber to selectively convey fluid from the chamber. The valve element is moveable between a first position allowing fluid from the source to be conveyed into the chamber and to the output channel, and a second position preventing the flow of fluid from the source to the output

[0013] In another embodiment, the proportional directional control valve includes a valve operating mechanism that permits communication between a first channel and a second channel through a linear motion of a valve element of the valve operating mechanism, a magnetic assembly operatively connected to the valve element where the magnetic assembly provides a magnetic field responsive to the linear motion of the valve element, a magnetic positioning sensor assembly configured to generate an electrical signal responsive to a change in the magnetic field, and a control system controlling the linear movement of the valve element in response to the electrical signal. The valve operating mechanism further includes at least one solenoid assembly electrically

channel.

coupled to the control system where a drive pin of the solenoid assembly is reciprocally moveable responsive to a current provided by the control system. The current provided is adjusted by the control system in response to the electrical signal. The magnetic positioning sensor assembly preferably includes a linear Hall-effect sensor where the electrical signal is an output voltage proportional to the change in the magnetic field

[0014] The control system includes a micro-controller mounted within an interior chamber of a first portion of a housing of the proportional directional control valve, an operator interface port coupled to the micro-controller, and a set-up device configured to provide values for operational parameter(s) of the proportional directional control valve. The control system also may include a wireless transceiver coupled to the micro-controller to receive from a wireless device, values for at least one operational parameter of the proportional directional control valve.

[0015] The proportional directional control valve also includes a main connector disposed at a second end of the first portion and coupled to the micro-controller where the main connector is configured to receive electrical power and to provide the electrical power to the micro-controller, to receive a command signal where the command signal is processed by the micro-controller to drive the solenoid assembly(s). The proportional directional control valve further includes an electronic potting material disposed on the micro-controller where the electronic potting material provides a protective barrier for the micro-controller.

[0016] The magnetic assembly includes a non-magnetic housing having a passageway extending axially through a portion of the non-magnetic housing, a first magnet disposed in the passageway, a second magnet disposed in the passageway where a south pole of the second magnet is separated from a north pole of the first magnet by a non-magnetic spacer in the passageway, and a retaining ring disposed in the passageway holding the first and second magnets and the non-magnetic spacer in a fixed position.

[0017] Additional aspects of the invention will be apparent to those of ordinary skill in the art in view of the detailed description of various embodiments, which is made with reference to the drawings, a brief description of which is provided below.

Brief Description of the Drawings

[0018] FIGURE 1 is a cross sectional view of the proportional directional control valve with a magnetic positioning sensor in accordance with an embodiment of the invention.

[0019] FIGURE 2 is an isometric view of the proportional directional control valve of FIG. 1.

[0020] FIGURE 3 is an isometric view of a proportional directional control valve with a magnetic positioning sensor in accordance with another embodiment of the invention.

[0021] FIGURE 4 is a cut away view of the proportional directional control valve of FIG. 1.

[0022] FIGURE 5 is a more detailed cross-sectional view of a magnetic assembly of the proportional directional control valve of FIG. 1.

[0023] FIGURE 6 is another isometric view of the proportional directional control valve of FIG. 1.

[0024] FIGURE 7 is an electrical schematic diagram of the proportional directional control valve of FIG. 1.

[0025] FIGURES 8-9 are an exemplary screen shot that may be displayed on a computer display screen coupled to the proportional directional control valve of FIG. 1.

Description of the Preferred Examples

[0026] The description of the preferred examples is to be construed as exemplary only and does not describe every possible embodiment of the invention.

Numerous alternative embodiments could be implemented, using either current technology or technology developed after the filing date of this patent, which would still fall within the scope of the claims defining the invention.

[0027] A proportional directional control valve with a magnetic positioning sensor is disclosed herein. In a preferred embodiment, the magnetic positioning sensor includes a linear Hall-effect sensor. Stated generally, the proportional directional control valve includes (a) a valve operating mechanism that permits communication between a first and a second channel through a linear motion of a valve element of the valve operating mechanism, (b) a magnetic assembly operatively connected to the valve element where a magnetic field of the

magnetic assembly is responsive to the linear motion of the valve element, (c) a magnetic positioning sensor assembly configured to generate an output electrical signal (e.g., an output voltage) responsive to a change in the magnetic field, and (d) a control system that controls the linear movement of the valve element in response to the electrical signal. The valve operating mechanism includes a solenoid assembly and valve assembly configured with a chamber having the first and second channel disposed therein. A source of fluid is selectively connected to the first channel. The linear motion of the valve element within the chamber of the valve assembly allows the fluid from the source to be selectively received by the first channel and to be selectively conveyed from the second channel.

electronic controller having a microprocessor and memory that controls the linear movement of a drive pin of the solenoid assembly in response to receipt of the output electrical signal from the Hall-effect sensor which is compared by the microprocessor to a command electrical input signal to produce the correct current to drive the solenoid(s). (i.e., in response to receipt of an output voltage directly proportional to the change in the magnetic field). The drive pin is moveably coupled to the valve element. The electronic controller converts the electrical signal into an associated current. The current causes linear motion, or movement, of the drive pin of the solenoid assembly coupled to the electronic controller, thereby changing a position of the valve element and the magnetic assembly. The change in the position of the valve element alters the fluid flow

out of the second channel. The change in the position of the magnetic assembly generates a magnetic field responsive to the linear motion of the valve element, and an associated new electrical signal is transmitted by the linear Hall-effect sensor to the electronic controller. Accordingly, precise position control, velocity control, speed control, force control or pressure control is provided to an apparatus coupled to the second channel of the proportional directional control valve with a magnetic positioning sensor.

[0029] An advantageous feature of the proportional directional control valve having a linear Hall-effect sensor in accordance with the invention is a substantial cost savings over prior art proportional directional control valves utilizing other types of valve element positioning sensors. For example, a typical linear Hall-effect sensor utilizing changes in a magnetic field to detect a valve element position change is ten times less costly than an LVDT utilizing changes in an inductance to detect a similar valve element position change.

[0030] Another advantageous feature of the proportional directional control valve having a linear Hall-effect sensor in accordance with the invention is that the onboard electronic controller, when coupled to another computer via one of any number of connecting protocols, can be easily programmed to set-up operational parameters of the proportional directional control valve, thereby precluding the need for manual pot adjustments and manual jumper reconfigurations for set-up and control of the proportional control valve.

[0031] FIG. 1 is a cross-sectional view of an embodiment of a proportional directional control valve 10 in accordance with the invention. Referring to FIG. 1,

the proportional directional control valve 10 includes a housing 12 having a first portion 14 with an interior chamber 18, and a second portion 16. The first housing portion 14 preferably comprises an extruded aluminum material. An electronic controller 20, preferably a microcontroller having a processor and a memory, is mounted within the interior chamber 18 of the first housing portion 14. An operator interface port 21 such as a universal serial bus (USB) port or DB9 is coupled to the electronic controller 20 at a first end of the first housing portion 14 and a main connector 23 is coupled to the electronic controller 20 at a second end of the first housing portion 14. Although the operator interface port 21 and the main connector 23 are located at opposite ends of the first housing portion 14 in the illustrated example, it is contemplated that they may be located in other suitable locations of the proportional directional control valve 10. [0032] A first and second solenoid assembly 22, 24 electrically coupled to the electronic controller 20 via their respective field windings, a valve assembly 26 operatively coupled to the first and second solenoid assembly 22, 24, and a magnetic assembly 28 moveably connected to the valve assembly 26, are disposed in the second housing portion 16. A magnetic positioning sensor assembly 30, also disposed in the second housing portion 16, is coupled to the electronic controller 20 via a coupling connector 32. FIG. 2 is an isometric view of the proportional directional control valve 10 illustrated in FIG. 1. [0033] Although the first and second solenoid assembly 22, 24 are illustrated in FIG. 1, it is contemplated that the proportional directional control valve 10 may

include only one solenoid assembly 22 (see, FIG. 3), depending on the application for which the proportional directional control valve 10 is being used. [0034] FIG. 4 is a cut away view of the proportional directional control valve 10. As illustrated in FIGs. 1 and 4, a chamber 82 is axially disposed in the valve assembly 26. The chamber has one input channel 36 coupled to a source of fluid (not separately illustrated), a first and second output channel 38, 40 to selectively convey fluid from the chamber, and a fluid reservoir 42. The fluid conveyed from the first and/or second output channel(s) 38, 40, is used to provide position control, velocity or speed control, force or pressure control, etc., depending on the required application. A position of a valve element 44, linearly moveable within the chamber of the valve assembly 26, determines the amount of fluid conveyed from the first and/or second output channel(s) 38, 40. For example, in a first position of the valve element 44, fluid from the source may be conveyed from the input channel 36 to the first output channel 38, while in a second position of the valve element 44, fluid from the source may be prevented from flowing to either the first or second output channels 38, 40. [0035] Although illustrated as a valve spool, it is contemplated that the valve element 44 may be any suitable valve element configured to operate in conjunction with the fluid channels and reservoir of the valve assembly 26 or configured to operate in conjunction with a variety connected loads (e.g., cylinder loads of an apparatus coupled to the proportional directional control valve 10) and their associated of flow rates. In addition, although configured with one input channel 36 and two output channels 38, 40, it is contemplated that any suitable

configuration of input channel(s) and output channel(s) may be utilized in the valve assembly 26.

[0036] Referring again to FIG. 1, the first solenoid assembly 22 includes a first solenoid armature 46 having a coil (not separately illustrated) disposed thereon. The first armature 46 is linearly moveable within a field winding 48 in response to a current provided by the electronic controller 20. The first solenoid assembly 22 also includes a first biasing pin 49 positioned between a first end of the first solenoid assembly 22 and a first end of the solenoid armature 46, and a first drive pin 52 fixedly coupled to a second end of the first solenoid armature 46 and operatively coupled to first end of the valve element 44. A first biasing spring 50 disposed on the first drive pin 52 is also included. Therefore, the first biasing pin 49, the first armature 46, and the first drive pin 52, together, are reciprocally moveable responsive to the current provided by the electronic controller 20. Further, when the current from the electronic controller 20 is applied to the field windings of the first solenoid assembly 22, the first drive pin 52 causes the valve element 44 to move to a new position, thereby increasing or decreasing the fluid flowing from the first and/or second output channels 38, 40. Thus, when the first solenoid armature 46 exerts a force greater then a predetermined force on the valve element 44, the first biasing spring 50 disengages from the first biasing pin 49 via a first mechanical stop (not separately illustrated) thereby allowing only the force greater than the predetermined force to engage the valve element 44. [0037] As previously mentioned, the magnetic assembly 28 is moveably connected to the valve assembly 26. Specifically, as illustrated in FIG. 1, a

second end of the valve element 44 is connected to a first end of the magnetic assembly 28 such that linear movement of the valve element 44 causes equal linear movement of the magnetic assembly 28, and vice versa.

[0038] FIG. 5 is a more detailed cross-sectional view of the magnetic assembly 28 illustrated in FIG. 1. The magnetic assembly 28 includes a non-magnetic housing 54 having a passageway extending axially through a portion of the nonmagnetic housing. Although preferably made of a beryllium copper material, the non-magnetic housing 54 may be made of one of any number of suitable nonmagnetic materials such as aluminum or stainless steel or titanium. A first magnet 56 is located in the first end of the passageway and a second magnet 58 is located in a second end of the passageway. The first and second magnets 56, 58 are preferably made of a samarium cobalt material. Although the south pole of the second magnet 58 is separated from a north pole of the first magnet 56 by a non-magnetic spacer 57 in the passageway, the first and second magnets 56, 58 may be configured in any manner with pole opposing each other. A retaining ring 60 holds the first and second magnets 56, 58 and the non-magnetic spacer 57 in a fixed position. Optional O-rings may also be included to center the first and second magnets 56, 58, thereby absorbing tolerance build up of individual components of the magnetic assembly 28.

[0039] Referring again to FIG. 1, the second solenoid assembly 24 includes a second solenoid armature 64 having a coil (not separately illustrated) disposed thereon. The second solenoid armature 64 is linearly moveable within a second field winding 66 in response to the current provided by the electronic controller

20. The second solenoid assembly 24 also includes a second biasing pin 68 positioned between a first end of the second solenoid assembly 24 and a second end of the second solenoid armature 64, and a second drive pin 72 fixedly coupled to a first end of the solenoid armature 64 and operatively coupled to a second end of the valve element 44. A second biasing spring 70 disposed on the second drive pin 72 is also included. Therefore, the second biasing pin 68, the second armature 64, and the second drive pin 72, together, are reciprocally moveable responsive to the current provided by the electronic controller 20. Further, when the current from the controller 20 is applied to the field windings of the second solenoid assembly 24, the second drive pin 72 causes the magnetic assembly 28 and therefore the valve element 44 to move to a new position. thereby increasing or decreasing the fluid flowing from the first and/or second output channels 38, 40. Thus, when the second solenoid armature 64 exerts a force greater than a predetermined force on the valve element 44 (by way of the magnetic assembly 28), the second biasing spring 70 disengages from the second biasing pin 68 via a second mechanical stop (not separately illustrated) thereby allowing only the force greater than the predetermined force to engage the valve element 44.

[0040] As previously mention, a magnetic positioning sensor 30 is coupled to the electronic controller 20 via the coupling connector 32. FIG. 6 is another isometric view of the proportional directional control valve 10. As illustrated in FIG. 6, the coupling connector 32 enables the connection between the magnetic positioning sensor 30 and the electronic controller 20 when the second housing portion 16 is

mated to the first housing portion 14. The coupling connector 32 preferably comprises a pinned terminal connector configured to transmit a magnetic positioning sensor output (*i.e.*, an output voltage) to the electronic controller 20 however other suitable coupling connectors (e.g., a small stereo pin or triaxial connector having a supply voltage input, a ground input and the magnetic positioning sensor output) may be utilized. In the illustrated example, the magnetic positioning sensor 30 preferably comprises a linear Hall-effect sensor such as the Ratiometric Linear Hall-Effect Sensor for High-Temperature Operation manufactured by Allegro Micro Systems, Inc., however other magnetic positioning sensors may be utilized.

[0041] Prior to operation of the proportional directional control valve 10, the magnetic positioning sensor 30, in this case a linear Hall-effect sensor, is calibrated in order to enable the electronic controller 20 to command operation of the first and second solenoid assemblies 22, 24 and therefore, shifting of the valve element 44. Calibration ensures proper tracking with a known command electrical signal received via the operator interface port 21. For example, if a command electrical signal is used to generate a +/- 10 volt to command a shift in the valve, at 0 volts the valve element 44 is centered, at +10 volts the second solenoid assembly 24 is fully shifted and at -10 volts the first solenoid assembly 22 is fully shifted in an opposite direction. Calibration of the magnetic positioning sensor 30 therefore includes aligning magnetic positioning sensor 30 output voltages associated with a centered valve element 44, a full shift of the second

solenoid assembly 24 and a full shift of the first solenoid assembly 22 with known command electrical signal values.

[0042] During operation of the proportional directional control valve 10, the magnetic positioning sensor 30 detects changes in the magnetic field resulting from the magnetic assembly 28 as the magnetic assembly 28 moves linearly with the valve element 44. At discrete time intervals, an output voltage proportional to a change in the magnetic field is generated by the linear Hall-effect sensor. The magnetic positioning sensor output voltage, received by the electronic controller 20, is then compared with the command electrical input signal to form an adjusting signal used to alter the current used to drive the first or second solenoid assembly 22, 24. In this way, the position of the valve element 44 can be adjusted, detected and utilized in a "feedback control" scheme to control fluid flow into and out of the valve assembly 26, thereby enabling precise position control, velocity or speed control, or force or pressure control.

[0043] The main connector 23 is preferably coupled to the electronic controller 20 at a second end of the first portion 14 of the housing. As illustrated by the schematic diagram of FIG. 7, the main connector 23 includes seven pins and is configured (1) to receive electrical power (e.g., 9 to 32 volts) and then to provide the electrical power to the electronic controller 20 and other components of the proportional directional control valve 10, (2) to receive a command signal (e.g., +/- 5 volts, +/-10 volts, 4-20 mA), the command signal being processed by the electronic controller to drive the solenoid assembly, (3) to provide an alarm output signal equal to an input power to power relay or lamp when open load is

detected, (4) to enable the current to the solenoid assemblies 22, 24 to be monitored, and (5) to enable overall operation of the proportional directional control valve 10 (e.g., contact closure to enable overall operation).

[0044] The operator interface port 21 is coupled to the electronic controller 20 to enable operator set-up of the operating parameters for the proportional directional control valve 10. In addition, an optional removable protective cap 80 may be included to protect the operator interface port 21 from dust and other contaminants while allowing operator access to the operator interface port 21 during parameter set-up. Although preferably located at the first end of the first portion 14, the operator interface port 21 and the removable protective cover 80 may be located in any suitable location of the proportional directional control valve 10.

[0045] Use of the operator interface port 21 enables communication between the electronic controller 20 and an appropriate communicating device coupled to the operator interface port 21. For example, the communicating devices may include personal computers (e.g., laptop, tablet, desktop computers) and handheld computers such as PDA's, communicating with the electronic controller 20 via a USB port, an RS-232 port, etc. Similarly, the communicating devices may include wireless devices such as mobile computers or mobile telephones, communicating with the electronic controller 20 via an IEEE 802.11 wireless transceiver, a Bluetooth™ wireless transceiver, a mobile station transceiver such as a code division multiple access mobile (CDMA) transceiver, to name a few.

[0046] Using the operator interface port 21 and an appropriate communicating device, an operator may configure parameters of the proportional directional control valve 10 to operate at within predetermined ranges. The operating parameters may include for example, a solenoid coil voltage, a command input voltage or a command input current, solenoid enable switch, a first ("A") and/or second ("B") solenoid null parameter, a first and/or second solenoid gain parameter, a first and/or second solenoid acceleration parameter, a first and/or second solenoid deceleration parameter, a dither frequency parameter and a first and/or second solenoid dither amplitude parameter. For example, utilizing a laptop computer operatively connected to the operator interface port 21, an operator may set-up the proportional directional control valve 10 with an operating coil voltage of 24 volts, a command voltage input of +/-10 volts, solenoid enable, a solenoid A and/or B null parameter of 0 to 1.000 amp, a solenoid A and/or B gain parameter of 0 to 2.500 amps, a solenoid A and/or B acceleration parameter of 0 to 30 seconds, a solenoid A and/or B deceleration parameter of 0 to 30 seconds, a dither frequency parameter of 0 to 360 Hz, and a solenoid A and/or B dither amplitude of 0 to 20%. FIGS. 8-9 illustrate an exemplary screen shot that may be displayed on a computer display screen coupled to the operator interface port 21 during parameter configuration of the proportional directional control valve 10.

[0047] In addition, the electronic controller 20 is mounted to an electronic board in the interior chamber 18 of the first housing portion 14. The electronic board, including the electronic controller 20 and associated electronics, is surrounded by

an electronic potting material (e.g., a silicon material) to provide a protective barrier against dust, fluids and other contaminants. Similarly, the potting material is also disposed on an interior portion of the main connector 23, an interior portion of the coupling connector 23 and an interior portion of the operator interface port 21.

[0048] As may be apparent from the discussion above, the "on-board" mounting of the electronic controller 20 and associated electronics in the first housing portion 14 enables the first housing portion 14 to be easily separated from the second housing portion 16 for maintenance purposes (see, FIG. 6). The onboard mounting of the electronic controller 20 and associated electronics also precludes the need for manual pot adjustments and manual jumper reconfigurations currently required for prior art proportional control valves having remotely located electronics. Further, use of the on-board electronic controller 20 and associated electronics enables self-teaching proportional-integralderivative (PID) control of the proportional directional control valve 10. [0049] As may also be apparent form the discussion above, use of the linear Halleffect sensor for detecting a change in the magnetic field by linear movement of the magnetic positioning sensor assembly 30 responsive to linear movement of the valve element 44, decreases the cost of the proportional directional control valve 10 as compared to other proportional directional control valves using other linear positioning sensing devices such as the LVDT.

[0050] From the foregoing, it will be observed that numerous variations and modifications may be affected without departing from the scope of the novel

concept of the invention. It is to be understood that no limitations with respect to the specific methods and apparatus illustrated herein is intended or should be inferred. It is, of course, intended to cover by the appended claims all such modifications as fall within the scope of the claims.